

VECTOR MECHANICS for ENGINEERS

STATICS | DYNAMICS

ELEVENTH EDITION

Beer • Johnston • Mazurek

Cornwell • Self

Mc
Graw
Hill
Education

Eleventh Edition

Vector Mechanics For Engineers

Statics and Dynamics

Ferdinand P. Beer

Late of Lehigh University

E. Russell Johnston, Jr.

Late of University of Connecticut

David F. Mazurek

U.S. Coast Guard Academy

Phillip J. Cornwell

Rose-Hulman Institute of Technology

Brian P. Self

California Polytechnic State University—San Luis Obispo





VECTOR MECHANICS FOR ENGINEERS: STATICS AND DYNAMICS, ELEVENTH EDITION

Published by McGraw-Hill Education, 2 Penn Plaza, New York, NY 10121. Copyright © 2016 by McGraw-Hill Education. All rights reserved. Printed in the United States of America. Previous editions © 2013, 2010, and 2007. No part of this publication may be reproduced or distributed in any form or by any means, or stored in a database or retrieval system, without the prior written consent of McGraw-Hill Education, including, but not limited to, in any network or other electronic storage or transmission, or broadcast for distance learning.

Some ancillaries, including electronic and print components, may not be available to customers outside the United States.

This book is printed on acid-free paper.

1 2 3 4 5 6 7 8 9 0 DOW/DOW 1 0 9 8 7 6 5

ISBN 978-0-07-339824-2

MHID 0-07-339824-1

Managing Director: *Thomas Timp*

Global Brand Manager: *Raghothaman Srinivasan*

Director of Development: *Rose Koos*

Product Developer: *Robin Reed*

Brand Manager: *Thomas Scaife, Ph.D.*

Digital Product Analyst: *Dan Wallace*

Editorial Coordinator: *Samantha Donisi-Hamm*

Marketing Manager: *Nick McFadden*

LearnSmart Product Developer: *Joan Weber*

Content Production Manager: *Linda Avenarius*

Content Project Managers: *Jolynn Kilburg and Lora Neyens*

Buyer: *Laura Fuller*

Designer: *Matthew Backhaus*

Content Licensing Specialist (Image): *Carrie Burger*

Typeface: *10/12 Times LT Std*

Printer: *R. R. Donnelley*

All credits appearing on page or at the end of the book are considered to be an extension of the copyright page.

Library of Congress Cataloging-in-Publication Data

Beer, Ferdinand P. (Ferdinand Pierre), 1915–2003.

Vector mechanics for engineers. Statics and dynamics / Ferdinand P. Beer, Late of Lehigh University, E. Russell Johnston, Jr., Late of University of Connecticut, David F. Mazurek, U.S. Coast Guard Academy, Phillip J. Cornwell, Rose-Hulman Institute of Technology; with the collaboration of Brian P. Self, California Polytechnic State University, San Luis Obispo.—Eleventh edition.

pages cm

Includes index.

ISBN 978-0-07-339824-2

1. Statics. 2. Dynamics. I. Johnston, E. Russell (Elwood Russell), 1925–2010. II. Mazurek, David F. (David Francis) III. Title. IV. Title: Statics and dynamics.

TA350.B3552 2016

620.1'054—dc23

2014041301

The Internet addresses listed in the text were accurate at the time of publication. The inclusion of a website does not indicate an endorsement by the authors or McGraw-Hill Education, and McGraw-Hill Education does not guarantee the accuracy of the information presented at these sites.

www.mhhe.com

About the Authors

Ferdinand P. Beer. Born in France and educated in France and Switzerland, Ferd received an M.S. degree from the Sorbonne and an Sc.D. degree in theoretical mechanics from the University of Geneva. He came to the United States after serving in the French army during the early part of World War II and taught for four years at Williams College in the Williams-MIT joint arts and engineering program. Following his service at Williams College, Ferd joined the faculty of Lehigh University where he taught for thirty-seven years. He held several positions, including University Distinguished Professor and chairman of the Department of Mechanical Engineering and Mechanics, and in 1995 Ferd was awarded an honorary Doctor of Engineering degree by Lehigh University.

E. Russell Johnston, Jr. Born in Philadelphia, Russ received a B.S. degree in civil engineering from the University of Delaware and an Sc.D. degree in the field of structural engineering from the Massachusetts Institute of Technology. He taught at Lehigh University and Worcester Polytechnic Institute before joining the faculty of the University of Connecticut where he held the position of chairman of the Department of Civil Engineering and taught for twenty-six years. In 1991 Russ received the Outstanding Civil Engineer Award from the Connecticut Section of the American Society of Civil Engineers.

David F. Mazurek. David holds a B.S. degree in ocean engineering and an M.S. degree in civil engineering from the Florida Institute of Technology and a Ph.D. degree in civil engineering from the University of Connecticut. He was employed by the Electric Boat Division of General Dynamics Corporation and taught at Lafayette College prior to joining the U.S. Coast Guard Academy, where he has been since 1990. He is a registered Professional Engineer in Connecticut and Pennsylvania, and has served on the American Railway Engineering & Maintenance-of-Way Association's Committee 15—Steel Structures since 1991. He is a Fellow of the American Society of Civil Engineers, and was elected to the Connecticut Academy of Science and Engineering in 2013. He was the 2014 recipient of both the Coast Guard Academy's Distinguished Faculty Award and its Center for Advanced Studies Excellence in Scholarship Award. Professional interests include bridge engineering, structural forensics, and blast-resistant design.

Phillip J. Cornwell. Phil holds a B.S. degree in mechanical engineering from Texas Tech University and M.A. and Ph.D. degrees in mechanical and aerospace engineering from Princeton University. He is currently a professor of mechanical engineering and Vice President of Academic Affairs at Rose-Hulman Institute of Technology where he has taught since 1989. Phil received an SAE Ralph R. Teetor Educational Award in 1992, the Dean's Outstanding Teacher Award at Rose-Hulman in 2000, and the Board of Trustees' Outstanding Scholar Award at Rose-Hulman in 2001. Phil was one of the developers of the Dynamics Concept Inventory.

Brian P. Self. Brian obtained his B.S. and M.S. degrees in Engineering Mechanics from Virginia Tech, and his Ph.D. in Bioengineering from the University of Utah. He worked in the Air Force Research Laboratories before teaching at the U.S. Air Force Academy for seven years. Brian has taught in the Mechanical Engineering Department at Cal Poly, San Luis Obispo since 2006. He has been very active in the American Society of Engineering Education, serving on its Board from 2008–2010. With a team of five, Brian developed the Dynamics Concept Inventory to help assess student conceptual understanding. His professional interests include educational research, aviation physiology, and biomechanics.

Brief Contents

- 1 Introduction 1
- 2 Statics of Particles 15
- 3 Rigid Bodies: Equivalent Systems of Forces 82
- 4 Equilibrium of Rigid Bodies 169
- 5 Distributed Forces: Centroids and Centers of Gravity 230
- 6 Analysis of Structures 297
- 7 Internal Forces and Moments 367
- 8 Friction 429
- 9 Distributed Forces: Moments of Inertia 485
- 10 Method of Virtual Work 573
- 11 Kinematics of Particles 615
- 12 Kinetics of Particles: Newton's Second Law 718
- 13 Kinetics of Particles: Energy and Momentum Methods 795
- 14 Systems of Particles 915
- 15 Kinematics of Rigid Bodies 977
- 16 Plane Motion of Rigid Bodies: Forces and Accelerations 1107
- 17 Plane Motion of Rigid Bodies: Energy and Momentum Methods 1181
- 18 Kinetics of Rigid Bodies in Three Dimensions 1264
- 19 Mechanical Vibrations 1332

Appendix: Fundamentals of Engineering Examination A1

Answers to Problems AN1

Photo Credits C1

Index I1

Contents

Preface	xi
Guided Tour	xv
Digital Resources	xviii
Acknowledgments	xx
List of Symbols	xxi

1 Introduction 1

- 1.1 What is Mechanics? 2
- 1.2 Fundamental Concepts and Principles 3
- 1.3 Systems of Units 5
- 1.4 Converting between Two Systems of Units 10
- 1.5 Method of Solving Problems 12
- 1.6 Numerical Accuracy 14

2 Statics of Particles 15

- 2.1 Addition of Planar Forces 16
- 2.2 Adding Forces by Components 29
- 2.3 Forces and Equilibrium in a Plane 39
- 2.4 Adding Forces in Space 52
- 2.5 Forces and Equilibrium in Space 66
- Review and Summary 75
- Review Problems 79

3 Rigid Bodies: Equivalent Systems of Forces 82

- 3.1 Forces and Moments 84
- 3.2 Moment of a Force about an Axis 105
- 3.3 Couples and Force-Couple Systems 120
- 3.4 Simplifying Systems of Forces 136
- Review and Summary 161
- Review Problems 166

4 Equilibrium of Rigid Bodies 169

- 4.1 Equilibrium in Two Dimensions 172
- 4.2 Two Special Cases 195
- 4.3 Equilibrium in Three Dimensions 204
- Review and Summary 225
- Review Problems 227

5 Distributed Forces: Centroids and Centers of Gravity 230

- 5.1 Planar Centers of Gravity and Centroids 232
- 5.2 Further Considerations of Centroids 249
- 5.3 Additional Applications of Centroids 262
- 5.4 Centers of Gravity and Centroids of Volumes 273
- Review and Summary 291
- Review Problems 295

6 Analysis of Structures 297

- 6.1 Analysis of Trusses 299
- 6.2 Other Truss Analyses 317
- 6.3 Frames 330
- 6.4 Machines 348
- Review and Summary 361
- Review Problems 364

7 Internal Forces and Moments 367

- 7.1 Internal Forces in Members 368
- 7.2 Beams 378
- 7.3 Relations Among Load, Shear, and Bending Moment 391
- *7.4 Cables 403
- *7.5 Catenary Cables 416
- Review and Summary 424
- Review Problems 427

8 Friction 429

- 8.1 The Laws of Dry Friction 431
- 8.2 Wedges and Screws 450
- *8.3 Friction on Axles, Disks, and Wheels 459
- 8.4 Belt Friction 469
- Review and Summary 479
- Review Problems 482

9 Distributed Forces: Moments of Inertia 485

- 9.1 Moments of Inertia of Areas 487
- 9.2 Parallel-Axis Theorem and Composite Areas 498
- *9.3 Transformation of Moments of Inertia 513
- *9.4 Mohr's Circle for Moments of Inertia 523
- 9.5 Mass Moments of Inertia 529
- *9.6 Additional Concepts of Mass Moments of Inertia 549
- Review and Summary 564
- Review Problems 570

10 Method of Virtual Work 573

- *10.1 The Basic Method 574
- *10.2 Work, Potential Energy, and Stability 595
- Review and Summary 609
- Review Problems 612

11 Kinematics of Particles 615

- 11.1 Rectilinear Motion of Particles 617
- 11.2 Special Cases and Relative Motion 635
- *11.3 Graphical Solutions 652
- 11.4 Curvilinear Motion of Particles 663
- 11.5 Non-Rectangular Components 690
- Review and Summary 711
- Review Problems 715

12 Kinetics of Particles: Newton's Second Law 718

- 12.1 Newton's Second Law and Linear Momentum 720
- 12.2 Angular Momentum and Orbital Motion 763
- *12.3 Applications of Central-Force Motion 774
 - Review and Summary 788
 - Review Problems 792

13 Kinetics of Particles: Energy and Momentum Methods 795

- 13.1 Work and Energy 797
- 13.2 Conservation of Energy 827
- 13.3 Impulse and Momentum 855
- 13.4 Impacts 877
 - Review and Summary 905
 - Review Problems 911

14 Systems of Particles 915

- 14.1 Applying Newton's Second Law and Momentum Principles to Systems of Particles 917
- 14.2 Energy and Momentum Methods for a System of Particles 936
- *14.3 Variable Systems of Particles 950
 - Review and Summary 970
 - Review Problems 974

15 Kinematics of Rigid Bodies 977

- 15.1 Translation and Fixed Axis Rotation 980
- 15.2 General Plane Motion: Velocity 997
- 15.3 Instantaneous Center of Rotation 1015
- 15.4 General Plane Motion: Acceleration 1029
- 15.5 Analyzing Motion with Respect to a Rotating Frame 1048
- *15.6 Motion of a Rigid Body in Space 1065
- *15.7 Motion Relative to a Moving Reference Frame 1082
 - Review and Summary 1097
 - Review Problems 1104

16 Plane Motion of Rigid Bodies: Forces and Accelerations 1107

- 16.1 Kinetics of a Rigid Body 1109
- 16.2 Constrained Plane Motion 1144
- Review and Summary 1085
- Review Problems 1087

17 Plane Motion of Rigid Bodies: Energy and Momentum Methods 1181

- 17.1 Energy Methods for a Rigid Body 1183
- 17.2 Momentum Methods for a Rigid Body 1211
- 17.3 Eccentric Impact 1234
- Review and Summary 1256
- Review Problems 1260

18 Kinetics of Rigid Bodies in Three Dimensions 1264

- 18.1 Energy and Momentum of a Rigid Body 1266
- *18.2 Motion of a Rigid Body in Three Dimensions 1285
- *18.3 Motion of a Gyroscope 1305
- Review and Summary 1323
- Review Problems 1328

19 Mechanical Vibrations 1332

- 19.1 Vibrations without Damping 1334
- 19.2 Free Vibrations of Rigid Bodies 1350
- 19.3 Applying the Principle of Conservation of Energy 1364
- 19.4 Forced Vibrations 1375
- 19.5 Damped Vibrations 1389
- Review and Summary 1403
- Review Problems 1408

Appendix: Fundamentals of Engineering Examination A1

Answers to Problems AN1

Photo Credits C1

Index I1

Preface

Objectives

A primary objective in a first course in mechanics is to help develop a student's ability first to analyze problems in a simple and logical manner, and then to apply basic principles to their solutions. A strong conceptual understanding of these basic mechanics principles is essential for successfully solving mechanics problems. We hope this text will help instructors achieve these goals.

General Approach

Vector algebra was introduced at the beginning of the first volume and is used in the presentation of the basic principles of statics, as well as in the solution of many problems, particularly three-dimensional problems. Similarly, the concept of vector differentiation will be introduced early in this volume, and vector analysis will be used throughout the presentation of dynamics. This approach leads to more concise derivations of the fundamental principles of mechanics. It also makes it possible to analyze many problems in kinematics and kinetics which could not be solved by scalar methods. The emphasis in this text, however, remains on the correct understanding of the principles of mechanics and on their application to the solution of engineering problems, and vector analysis is presented chiefly as a convenient tool.[†]

Practical Applications Are Introduced Early. One of the characteristics of the approach used in this book is that mechanics of *particles* is clearly separated from the mechanics of *rigid bodies*. This approach makes it possible to consider simple practical applications at an early stage and to postpone the introduction of the more difficult concepts. For example:

- In *Statics*, the statics of particles is treated first, and the principle of equilibrium of a particle was immediately applied to practical situations involving only concurrent forces. The statics of rigid bodies is considered later, at which time the vector and scalar products of two vectors were introduced and used to define the moment of a force about a point and about an axis.
- In *Dynamics*, the same division is observed. The basic concepts of force, mass, and acceleration, of work and energy, and of impulse and momentum are introduced and first applied to problems involving only particles. Thus, students can familiarize themselves with the three basic methods used in dynamics and learn their respective advantages before facing the difficulties associated with the motion of rigid bodies.

[†]In a parallel text, *Mechanics for Engineers*, fifth edition, the use of vector algebra is limited to the addition and subtraction of vectors, and vector differentiation is omitted.

NEW!

The 11th edition has undergone a complete rewrite to modernize and streamline the language throughout the text.

2.2 ADDING FORCES BY COMPONENTS

In Sec. 2.1E, we described how to resolve a force into components. Here we discuss how to add forces by using their components, especially rectangular components. This method is often the most convenient way to add forces and, in practice, is the most common approach. (Note that we can readily extend the properties of vectors established in this section to the rectangular components of any vector quantity, such as velocity or momentum.)

2.2A Rectangular Components of a Force: Unit Vectors

In many problems, it is useful to resolve a force into two components that are perpendicular to each other. Figure 2.14 shows a force F resolved into a component F_x along the x axis and a component F_y along the y axis. The parallelogram drawn to obtain the two components is a rectangle, and F_x and F_y are called **rectangular components**.

The x and y axes are usually chosen to be horizontal and vertical, respectively, as in Fig. 2.14; they may, however, be chosen in any two perpendicular directions, as shown in Fig. 2.15. In determining the

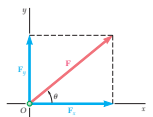


Fig. 2.14 Rectangular components of a force F .

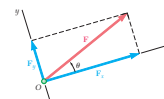


Fig. 2.15 Rectangular components of a force F for axes rotated away from horizontal and vertical.

17.1 ENERGY METHODS FOR A RIGID BODY

We now use the principle of work and energy to analyze the plane motion of rigid bodies. As we pointed out in Chap. 13, the method of work and energy is particularly well adapted to solving problems involving velocities and displacements. Its main advantage is that the work of forces and the kinetic energy of particles are scalar quantities.

17.1A Principle of Work and Energy

To apply the principle of work and energy to the motion of a rigid body, we again assume that the rigid body is made up of a large number n of particles of mass Δm_i . From Eq. (14.30) of Sec. 14.2B, we have

Principle of work and energy, rigid body

$$T_1 + U_{1 \rightarrow 2} = T_2 \quad (17.1)$$

where T_1 , T_2 = the initial and final values of total kinetic energy of particles forming the rigid body

$U_{1 \rightarrow 2}$ = work of all forces acting on various particles of the body

Just as we did in Chap. 13, we can express the work done by nonconservative forces as $U_{1 \rightarrow 2}^{NC}$, and we can define potential energy terms for conservative forces. Then we can express Eq. (17.1) as

$$T_1 + V_{g_1} + V_{e_1} + U_{1 \rightarrow 2}^{NC} = T_2 + V_{g_2} + V_{e_2} \quad (17.1')$$

where V_{g_1} and V_{g_2} are the initial and final gravitational potential energy of the center of mass of the rigid body with respect to a reference point or datum, and V_{e_1} and V_{e_2} are the initial and final values of the elastic energy associated with springs in the system.

We obtain the total kinetic energy

$$T = \frac{1}{2} \sum_{i=1}^n \Delta m_i v_i^2 \quad (17.2)$$

by adding positive scalar quantities, so it is itself a positive scalar quantity. You will see later how to determine T for various types of motion of a rigid body.

The expression $U_{1 \rightarrow 2}$ in Eq. (17.1) represents the work of all the forces acting on the various particles of the body whether these forces are internal or external. However, the total work of the internal forces holding together the particles of a rigid body is zero. To see this, consider two particles A and B of a rigid body and the two equal and opposite forces \mathbf{F} and $-\mathbf{F}$ they exert on each other (Fig. 17.1). Although, in general, small displacements $d\mathbf{r}$ and $d\mathbf{r}'$ of the two particles are different, the components of these displacements along AB must be equal; otherwise, the particles would not remain at the same distance from each other and the body would not be rigid. Therefore, the work of \mathbf{F} is equal in magnitude and

New Concepts Are Introduced in Simple Terms. Since this text is designed for the first course in dynamics, new concepts are presented in simple terms and every step is explained in detail. On the other hand, by discussing the broader aspects of the problems considered, and by stressing methods of general applicability, a definite maturity of approach has been achieved. For example, the concept of potential energy is discussed in the general case of a conservative force. Also, the study of the plane motion of rigid bodies is designed to lead naturally to the study of their general motion in space. This is true in kinematics as well as in kinetics, where the principle of equivalence of external and effective forces is applied directly to the analysis of plane motion, thus facilitating the transition to the study of three-dimensional motion.

Fundamental Principles Are Placed in the Context of Simple Applications. The fact that mechanics is essentially a *deductive* science based on a few fundamental principles is stressed. Derivations have been presented in their logical sequence and with all the rigor warranted at this level. However, the learning process being largely *inductive*, simple applications are considered first. For example:

- The statics of particles precedes the statics of rigid bodies, and problems involving internal forces are postponed until Chap. 6.
- In Chap. 4, equilibrium problems involving only coplanar forces are considered first and solved by ordinary algebra, while problems involving three-dimensional forces and requiring the full use of vector algebra are discussed in the second part of the chapter.
- The kinematics of particles (Chap. 11) precedes the kinematics of rigid bodies (Chap. 15).
- The fundamental principles of the kinetics of rigid bodies are first applied to the solution of two-dimensional problems (Chaps. 16 and 17), which can be more easily visualized by the student, while three-dimensional problems are postponed until Chap. 18.

The Presentation of the Principles of Kinetics Is Unified.

The eleventh edition of *Vector Mechanics for Engineers* retains the unified presentation of the principles of kinetics which characterized the previous ten editions. The concepts of linear and angular momentum are introduced in Chap. 12 so that Newton's second law of motion can be presented not only in its conventional form $\mathbf{F} = m\mathbf{a}$, but also as a law relating, respectively, the sum of the forces acting on a particle and the sum of their moments to the rates of change of the linear and angular momentum of the particle. This makes possible an earlier introduction of the principle of conservation of angular momentum and a more meaningful discussion of the motion of a particle under a central force (Sec. 12.3A). More importantly, this approach can be readily extended to the study of the motion of a system of particles (Chap. 14) and leads to a more concise and unified treatment of the kinetics of rigid bodies in two and three dimensions (Chaps. 16 through 18).

Systematic Problem-Solving Approach. New to this edition of the text, all the sample problems are solved using the steps of *Strategy*, *Modeling*, *Analysis*, and *Reflect & Think*, or the "SMART" approach.

NEW!

This methodology is intended to give students confidence when approaching new problems, and students are encouraged to apply this approach in the solution of all assigned problems.

Free-Body Diagrams Are Used Both to Solve Equilibrium Problems and to Express the Equivalence of Force Systems.

Free-body diagrams were introduced early in statics, and their importance was emphasized throughout. They were used not only to solve equilibrium problems but also to express the equivalence of two systems of forces or, more generally, of two systems of vectors. In dynamics we will introduce a kinetic diagram, which is a pictorial representation of inertia terms. The advantage of this approach becomes apparent in the study of the dynamics of rigid bodies, where it is used to solve three-dimensional as well as two-dimensional problems. By placing the emphasis on the free-body diagram and kinetic diagram, rather than on the standard algebraic equations of motion, a more intuitive and more complete understanding of the fundamental principles of dynamics can be achieved. This approach, which was first introduced in 1962 in the first edition of *Vector Mechanics for Engineers*, has now gained wide acceptance among mechanics teachers in this country. It is, therefore, used in preference to the method of dynamic equilibrium and to the equations of motion in the solution of all sample problems in this book.

A Careful Balance between SI and U.S. Customary Units Is Consistently Maintained.

Because of the current trend in the American government and industry to adopt the international system of units (SI metric units), the SI units most frequently used in mechanics are introduced in Chap. 1 and are used throughout the text. Approximately half of the sample problems and 60 percent of the homework problems are stated in these units, while the remainder are in U.S. customary units. The authors believe that this approach will best serve the need of the students, who, as engineers, will have to be conversant with both systems of units.

It also should be recognized that using both SI and U.S. customary units entails more than the use of conversion factors. Since the SI system of units is an absolute system based on the units of time, length, and mass, whereas the U.S. customary system is a gravitational system based on the units of time, length, and force, different approaches are required for the solution of many problems. For example, when SI units are used, a body is generally specified by its mass expressed in kilograms; in most problems of statics it will be necessary to determine the weight of the body in newtons, and an additional calculation will be required for this purpose. On the other hand, when U.S. customary units are used, a body is specified by its weight in pounds and, in dynamics problems, an additional calculation will be required to determine its mass in slugs (or $\text{lb}\cdot\text{s}^2/\text{ft}$). The authors, therefore, believe that problem assignments should include both systems of units.

The *Instructor's and Solutions Manual* provides six different lists of assignments so that an equal number of problems stated in SI units and in U.S. customary units can be selected. If so desired, two complete lists of assignments can also be selected with up to 75 percent of the problems stated in SI units.

Sample Problem 3.10

Three cables are attached to a bracket as shown. Replace the forces exerted by the cables with an equivalent force-couple system at A.

STRATEGY: First determine the relative position vectors drawn from point A to the points of application of the various forces and resolve the forces into rectangular components. Then sum the forces and moments.

MODELING and ANALYSIS: Note that $F_3 = (700 \text{ N})\mathbf{A}_{AC}$ where $\mathbf{A}_{AC} = \frac{\overline{AC}}{AC} = \frac{75\mathbf{i} - 150\mathbf{j} + 50\mathbf{k}}{175}$

Using meters and newtons, the position and force vectors are

$r_{AB} = \overline{AB} = 0.075\mathbf{i} + 0.050\mathbf{k}$	$F_B = 300\mathbf{i} - 600\mathbf{j} + 200\mathbf{k}$
$r_{AC} = \overline{AC} = 0.075\mathbf{i} - 0.050\mathbf{k}$	$F_C = 707\mathbf{i} - 707\mathbf{k}$
$r_{AD} = \overline{AD} = 0.100\mathbf{i} - 0.100\mathbf{j}$	$F_D = 600\mathbf{i} + 1039\mathbf{j}$

The force-couple system at A equivalent to the given forces consists of a force $\mathbf{R} = \Sigma \mathbf{F}$ and a couple $\mathbf{M}_A^R = \Sigma (\mathbf{r} \times \mathbf{F})$. Obtain the force \mathbf{R} by adding respectively the x , y , and z components of the forces:

$$\mathbf{R} = \Sigma \mathbf{F} = (1607 \text{ N}\mathbf{i} + 439 \text{ N}\mathbf{j}) = (507 \text{ N})\mathbf{k} \quad \leftarrow$$

(continued)

Fig. 3 Point of application of single tugboat to create same effect as given force system.

Remark: Since all the forces are contained in the plane of the figure, you would expect the sum of their moments to be perpendicular to that plane. Note that you could obtain the moment of each force component directly from the diagram by first forming the product of its magnitude and perpendicular distance to O and then assigning to this product a positive or a negative sign, depending upon the sense of the moment.

b. Single Tugboat. The force exerted by a single tugboat must be equal to \mathbf{R} , and its point of application A must be such that the moment of \mathbf{R} about O is equal to \mathbf{M}_O^R (Fig. 3). Observing that the position vector of A is $\mathbf{r} = x\mathbf{i} + 70\mathbf{j}$

you have

$$\mathbf{r} \times \mathbf{R} = \mathbf{M}_O^R$$

$$(x + 70\mathbf{j}) \times (9.04\mathbf{i} - 9.79\mathbf{j}) = -103\mathbf{k}$$

$$-(9.79\mathbf{k}) - 63\mathbf{k} = -103\mathbf{k} \quad x = 41.1 \text{ ft} \quad \leftarrow$$

REFLECT and THINK: Reducing the given situation to that of a single force makes it easier to visualize the overall effect of the tugboats in maneuvering the ocean liner. But in practical terms, having four boats applying force allows for greater control in slowing and turning a large ship in a crowded harbor.

Optional Sections Offer Advanced or Specialty Topics.

A large number of optional sections have been included. These sections are indicated by asterisks and thus are easily distinguished from those which form the core of the basic dynamics course. They can be omitted without prejudice to the understanding of the rest of the text.

The topics covered in the optional sections in statics include the reduction of a system of forces of a wrench, applications to hydrostatics, equilibrium of cables, products of inertia and Mohr's circle, the determination of the principal axes and the mass moments of inertia of a body of arbitrary shape, and the method of virtual work. The sections on the inertia properties of three-dimensional bodies are primarily intended for students who will later study in dynamics the three-dimensional motion of rigid bodies.

The topics covered in the optional sections in dynamics include graphical methods for the solution of rectilinear-motion problems, the trajectory of a particle under a central force, the deflection of fluid streams, problems involving jet and rocket propulsion, the kinematics and kinetics of rigid bodies in three dimensions, damped mechanical vibrations, and electrical analogues. These topics will be found of particular interest when dynamics is taught in the junior year.

The material presented in the text and most of the problems require no previous mathematical knowledge beyond algebra, trigonometry, elementary calculus, and the elements of vector algebra presented in Chaps. 2 and 3 of the volume on statics. However, special problems are included, which make use of a more advanced knowledge of calculus, and certain sections, such as Secs. 19.5A and 19.5B on damped vibrations, should be assigned only if students possess the proper mathematical background. In portions of the text using elementary calculus, a greater emphasis is placed on the correct understanding and application of the concepts of differentiation and integration, than on the nimble manipulation of mathematical formulas. In this connection, it should be mentioned that the determination of the centroids of composite areas precedes the calculation of centroids by integration, thus making it possible to establish the concept of moment of area firmly before introducing the use of integration.

Guided Tour

Chapter Introduction. Each chapter begins with a list of learning objectives and an outline that previews chapter topics. An introductory section describes the material to be covered in simple terms, and how it will be applied to the solution of engineering problems.

Chapter Lessons. The body of the text is divided into sections, each consisting of one or more sub-sections, several sample problems, and a large number of end-of-section problems for students to solve. Each section corresponds to a well-defined topic and generally can be covered in one lesson. In a number of cases, however, the instructor will find it desirable to devote more than one lesson to a given topic. *The Instructor's and Solutions Manual* contains suggestions on the coverage of each lesson.

Sample Problems. The Sample Problems are set up in much the same form that students will use when solving assigned problems, and they employ the SMART problem-solving methodology that students are encouraged to use in the solution of their assigned problems. They thus serve the double purpose of reinforcing the text and demonstrating the type of neat and orderly work that students should cultivate in their own solutions. In addition, in-problem references and captions have been added to the sample problem figures for contextual linkage to the step-by-step solution.

Concept Applications. Concept Applications are used within selected theory sections in the Statics volume to amplify certain topics, and they are designed to reinforce the specific material being presented and facilitate its understanding.

Solving Problems on Your Own. A section entitled *Solving Problems on Your Own* is included for each lesson, between the sample problems and the problems to be assigned. The purpose of these sections is to help students organize in their own minds the preceding theory of the text and the solution methods of the sample problems so that they can more successfully solve the homework problems. Also included in these sections are specific suggestions and strategies that will enable the students to more efficiently attack any assigned problems.

Homework Problem Sets. Most of the problems are of a practical nature and should appeal to engineering students. They are primarily designed, however, to illustrate the material presented in the text and to help students understand the principles of mechanics. The problems are grouped according to the portions of material they illustrate and, in general, are arranged in order of increasing difficulty. Problems requiring special attention are indicated by asterisks. Answers to 70 percent of the problems are given at the end of the book. Problems for which the answers are given are set in straight type in the text, while problems for which no answer is given are set in italic and red font color.



1

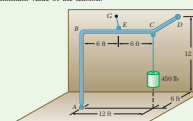
Introduction

The tallest skyscraper in the Western Hemisphere, One World Trade Center is a prominent feature of the New York City skyline. From its foundation to its structural components and mechanical systems, the design and operation of the tower is based on the fundamentals of engineering mechanics.

NEW!

Sample Problem 4.10

A 450-lb load hangs from the corner C of a rigid piece of pipe $ABCD$ that has been bent as shown. The pipe is supported by ball-and-socket joints A and D , which are fastened, respectively, to the floor and to a vertical wall, and by a cable attached at the midpoint E of the portion BC of the pipe and at a point G on the wall. Determine (a) where G should be located if the tension in the cable is to be minimum, (b) the corresponding minimum value of the tension.



STRATEGY: Draw the free-body diagram of the pipe showing the reactions at A and D . Isolate the unknown tension T and the known weight W by summing moments about the diagonal line AD , and compute values from the equilibrium equations.

MODELING and ANALYSIS:

Free-Body Diagram. The free-body diagram of the pipe includes the load $W = (-450 \text{ lb})\mathbf{j}$, the reactions at A and D , and the force T exerted by the cable (Fig. 1). To eliminate the reactions at A and D from the computations, take the sum of the moments of the forces about the line AD and set it equal to zero. Denote the unit vector along AD by λ , which enables you to write

$$\sum M_{AD} = 0: \lambda \cdot (\lambda \mathbf{E} \times \mathbf{T}) + \lambda \cdot (\lambda \mathbf{E} \times \mathbf{W}) = 0 \quad (1)$$

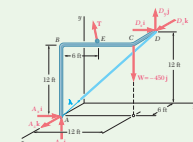


Fig. 1 Free-body diagram of pipe.

NEW!

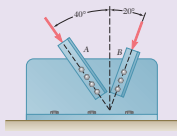
Over 650 of the homework problems in the text are new or revised.

Chapter Review and Summary. Each chapter ends with a review and summary of the material covered in that chapter. Marginal notes are used to help students organize their review work, and cross-references have been included to help them find the portions of material requiring their special attention.

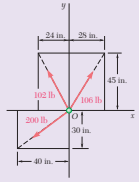
Review Problems. A set of review problems is included at the end of each chapter. These problems provide students further opportunity to apply the most important concepts introduced in the chapter.

Review Problems

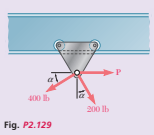
2.127 Two structural members *A* and *B* are bolted to a bracket as shown. Knowing that both members are in compression and that the force is 15 kN in member *A* and 10 kN in member *B*, determine by trigonometry the magnitude and direction of the resultant of the forces applied to the bracket by members *A* and *B*.



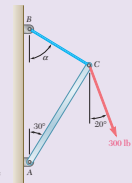
2.128 Determine the *x* and *y* components of each of the forces shown.



2.129 A hoist trolley is subjected to the three forces shown. Knowing that $\alpha = 40^\circ$, determine (a) the required magnitude of the force *P* if the resultant of the three forces is to be vertical, (b) the corresponding magnitude of the resultant.



2.130 Knowing that $\alpha = 55^\circ$ and that boom *AC* exerts on pin *C* a force directed along line *AC*, determine (a) the magnitude of that force, (b) the tension in cable *BC*.

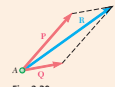


79

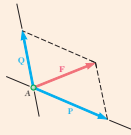
Review and Summary

In this chapter, we have studied the effect of forces on particles, i.e., on bodies of such shape and size that we may assume all forces acting on them apply at the same point.

Resultant of Two Forces
 Forces are *vector quantities*; they are characterized by a point of application, a magnitude, and a direction, and they add according to the parallelogram law (Fig. 2.30). We can determine the magnitude and direction of the resultant *R* of two forces *P* and *Q* either graphically or by trigonometry using the law of cosines and the law of sines (Sample Prob. 2.1).



Components of a Force
 Any given force acting on a particle can be resolved into two or more components, i.e., it can be replaced by two or more forces that have the same effect on the particle. A force *F* can be resolved into two components *P* and *Q* by drawing a parallelogram with *F* for its diagonal; the components *P* and *Q* are then represented by the two adjacent sides of the parallelogram (Fig. 2.31). Again, we can determine the components either graphically or by trigonometry [Sec. 2.1E].



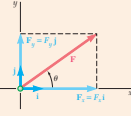
Rectangular Components; Unit Vectors
 A force *F* is resolved into two rectangular components if its components *F_x* and *F_y* are perpendicular to each other and are directed along the coordinate axes (Fig. 2.32). Introducing the unit vectors *i* and *j* along the *x* and *y* axes, respectively, we can write the components and the vector as [Sec. 2.2A]

$$F_x = F_x \mathbf{i} \quad F_y = F_y \mathbf{j} \quad (2.6)$$

and

$$\mathbf{F} = F_x \mathbf{i} + F_y \mathbf{j} \quad (2.7)$$

where *F_x* and *F_y* are the *scalar components* of *F*. These components, which can be positive or negative, are defined by the relations

$$F_x = F \cos \theta \quad F_y = F \sin \theta \quad (2.8)$$


75

Computer Problems. Accessible through Connect are problem sets for each chapter that are designed to be solved with computational software. Many of these problems are relevant to the design process; they may involve the analysis of a structure for various configurations and loadings of the structure, or the determination of the equilibrium positions of a given mechanism that may require an iterative method of solution. Developing the algorithm required to solve a given mechanics problem will benefit the students in two different ways: (1) it will help them gain a better understanding of the mechanics principles involved; (2) it will provide them with an opportunity to apply their computer skills to the solution of a meaningful engineering problem.

Concept Questions. Educational research has shown that students can often choose appropriate equations and solve algorithmic problems without having a strong conceptual understanding of mechanics principles.[†] To help assess and develop student conceptual understanding, we have included Concept Questions, which are multiple choice problems that require few, if any, calculations. Each possible incorrect answer typically represents a common misconception (e.g., students often think that a vehicle moving in a curved path at constant speed has zero acceleration). Students are encouraged to solve these problems using the principles and techniques discussed in the text and to use these principles to help them develop their intuition. Mastery and discussion of these Concept Questions will deepen students' conceptual understanding and help them to solve dynamics problems.

Free Body and Impulse-Momentum Diagram Practice Problems. Drawing diagrams correctly is a critical step in solving kinetics problems in dynamics. A new type of problem has been added to the text to emphasize the importance of drawing these diagrams. In Chaps. 12 and 16 the Free Body Practice Problems require students to draw a free-body diagram (FBD) showing the applied forces and an equivalent diagram called a “kinetic diagram” (KD) showing ma or its components and $\bar{I}\alpha$. These diagrams provide students with a pictorial representation of Newton's second law and are critical in helping students to correctly solve kinetic problems. In Chaps. 13 and 17 the Impulse-Momentum Diagram Practice Problems require students to draw diagrams showing the momenta of the bodies before impact, the impulses exerted on the body during impact, and the final momenta of the bodies. The answers to all of these questions can be accessed through Connect.

FREE-BODY PRACTICE PROBLEMS

16.F1 A 6-ft board is placed in a truck with one end resting against a block secured to the floor and the other leaning against a vertical partition. Draw the FBD and KD necessary to determine the maximum allowable acceleration of the truck if the board is to remain in the position shown.

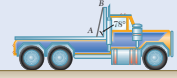


Fig. P16.F1

16.F2 A uniform circular plate of mass 3 kg is attached to two links AC and BD of the same length. Knowing that the plate is released from rest in the position shown, in which lines joining G to A and B are, respectively, horizontal and vertical, draw the FBD and KD for the plate.

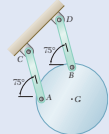


Fig. P16.F2

16.F3 Two uniform disks and two cylinders are assembled as indicated. Disk A weighs 20 lb and disk B weighs 12 lb. Knowing that the system is released from rest, draw the FBD and KD for the whole system.

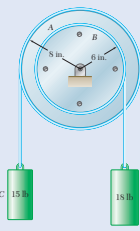


Fig. P16.F3

16.F4 The 400-lb crate shown is lowered by means of two overhead cranes. Knowing the tension in each cable, draw the FBD and KD that can be used to determine the angular acceleration of the crate and the acceleration of the center of gravity.

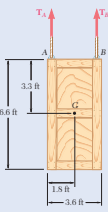


Fig. P16.F4

1039

[†]Hestenes, D., Wells, M., and Swakhamer, G (1992). The force concept inventory. *The Physics Teacher*, 30: 141–158.

Streveler, R. A., Litzinger, T. A., Miller, R. L., and Steif, P. S. (2008). Learning conceptual knowledge in the engineering sciences: Overview and future research directions, *JEE*, 279–294.

Digital Resources



Connect[®] Engineering provides online presentation, assignment, and assessment solutions. It connects your students with the tools and resources they'll need to achieve success. With Connect Engineering you can deliver assignments, quizzes, and tests online. A robust set of questions and activities are presented and aligned with the textbook's learning outcomes. As an instructor, you can edit existing questions and author entirely new problems. Integrate grade reports easily with Learning Management Systems (LMS), such as WebCT and Blackboard—and much more. Connect Engineering also provides students with 24/7 online access to a media-rich eBook, allowing seamless integration of text, media, and assessments. **To learn more, visit connect.mheducation.com**

Find the following instructor resources available through Connect:

- **Instructor's and Solutions Manual.** *The Instructor's and Solutions Manual* that accompanies the eleventh edition features solutions to all end of chapter problems. This manual also features a number of tables designed to assist instructors in creating a schedule of assignments for their course. The various topics covered in the text have been listed in Table I and a suggested number of periods to be spent on each topic has been indicated. Table II prepares a brief description of all groups of problems and a classification of the problems in each group according to the units used. Sample lesson schedules are shown in Tables III, IV, and V, together with various alternative lists of assigned homework problems.
- **Lecture PowerPoint Slides** for each chapter that can be modified. These generally have an introductory application slide, animated worked-out problems that you can do in class with your students, concept questions, and “what-if?” questions at the end of the units.
- **Textbook images**
- **Computer Problem sets** for each chapter that are designed to be solved with computational software.
- **C.O.S.M.O.S.**, the Complete Online Solutions Manual Organization System that allows instructors to create custom homework, quizzes, and tests using end-of-chapter problems from the text.

NEW!



LEARNSMART[®] LearnSmart is available as an integrated feature of McGraw-Hill Connect. It is an adaptive learning system designed to help students learn faster, study more efficiently, and retain more knowledge for greater success. LearnSmart assesses a student's knowledge of course content through a series of adaptive questions. It pinpoints concepts the student does not understand and maps out a personalized study plan for success. This innovative study tool also has features that allow instructors to see exactly what students have accomplished and a built-in assessment tool for graded assignments.



SMARTBOOK™ SmartBook™ is the first and only adaptive reading experience available for the higher education market. Powered by an intelligent diagnostic and adaptive engine, SmartBook facilitates the reading process by identifying what content a student knows and doesn't know through adaptive assessments. As the student reads, the reading material constantly adapts to ensure the student is focused on the content he or she needs the most to close any knowledge gaps.

Visit the following site for a demonstration of LearnSmart or Smart-Book: www.learnsmartadvantage.com

CourseSmart. This text is offered through CourseSmart for both instructors and students. CourseSmart is an online browser where students can purchase access to this and other McGraw-Hill textbooks in a digital format. Through their browser, students can access the complete text online at almost half the cost of a traditional text. Purchasing the eTextbook also allows students to take advantage of CourseSmart's web tools for learning, which include full text search, notes and highlighting, and e-mail tools for sharing notes among classmates. To learn more about CourseSmart options, contact your sales representative or visit **www.coursesmart.com**.

**NEW!**

Acknowledgments

A special thanks to our colleagues who thoroughly checked the solutions and answers to all problems in this edition and then prepared the solutions for the accompanying *Instructor's and Solutions Manual*, James Widmann of California Polytechnic State University and Amy Mazurek.

The authors thank the many companies that provided photographs for this edition.

We are pleased to acknowledge David Chelton, who carefully reviewed the entire text and provided many helpful suggestions for revising this edition.

The authors also thank the members of the staff at McGraw-Hill for their support and dedication during the preparation of this new edition. We particularly wish to acknowledge the contributions of Global Brand Manager Raghu Srinivasan, Brand Manager Thomas Scaife, Product Developers Robin Reed & Joan Weber, Content Project Manager Jolynn Kilburg, and Program Manager Lora Neyens.

David F. Mazurek
Phillip J. Cornwell
Brian P. Self

The authors gratefully acknowledge the many helpful comments and suggestions offered by focus group attendees and by users of the previous editions of *Vector Mechanics for Engineers*:

George Adams
Northeastern University
William Altenhof
University of Windsor
Sean B. Anderson
Boston University
Manohar Arora
Colorado School of Mines
Gilbert Baladi
Michigan State University
Francois Barthelat
McGill University
Oscar Barton, Jr.
U.S. Naval Academy
M. Asghar Bhatti
University of Iowa
Shaohong Cheng
University of Windsor
Philip Datsaris
University of Rhode Island
Timothy A. Doughty
University of Portland

Howard Epstein
University of Connecticut
Asad Esmaeily
Kansas State University, Civil
Engineering Department
David Fleming
Florida Institute of Technology
Jeff Hanson
Texas Tech University
David A. Jenkins
University of Florida
Shaofan Li
University of California, Berkeley
William R. Murray
Cal Poly State University
Eric Musselman
University of Minnesota, Duluth
Masoud Olia
Wentworth Institute of Technology
Renee K. B. Petersen
Washington State University

Amir G Rezaei
California State Polytechnic
University, Pomona
Martin Sadd
University of Rhode Island
Stefan Seelecke
North Carolina State University
Yixin Shao
McGill University
Muhammad Sharif
The University of Alabama
Anthony Sinclair
University of Toronto
Lizhi Sun
University of California, Irvine
Jeffrey Thomas
Northwestern University
Jiashi Yang
University of Nebraska
Xiangwa Zeng
Case Western Reserve University

List of Symbols

\mathbf{a}, a	Acceleration	\bar{k}	Centroidal radius of gyration
a	Constant; radius; distance; semimajor axis of ellipse	l	Length
$\bar{\mathbf{a}}, \bar{a}$	Acceleration of mass center	\mathbf{L}	Linear momentum
$\mathbf{a}_{B/A}$	Acceleration of B relative to frame in translation with A	L	Length; inductance
$\mathbf{a}_{P/\mathcal{F}}$	Acceleration of P relative to rotating frame \mathcal{F}	m	Mass
\mathbf{a}_c	Coriolis acceleration	m'	Mass per unit length
$\mathbf{A}, \mathbf{B}, \mathbf{C}, \dots$	Reactions at supports and connections	\mathbf{M}	Couple; moment
A, B, C, \dots	Points	\mathbf{M}_O	Moment about point O
A	Area	\mathbf{M}_O^R	Moment resultant about point O
b	Width; distance; semiminor axis of ellipse	M	Magnitude of couple or moment; mass of earth
c	Constant; coefficient of viscous damping	M_{OL}	Moment about axis OL
C	Centroid; instantaneous center of rotation; capacitance	n	Normal direction
d	Distance	\mathbf{N}	Normal component of reaction
$\mathbf{e}_n, \mathbf{e}_t$	Unit vectors along normal and tangent	O	Origin of coordinates
$\mathbf{e}_r, \mathbf{e}_\theta$	Unit vectors in radial and transverse directions	\mathbf{P}	Force; vector
e	Coefficient of restitution; base of natural logarithms	$\dot{\mathbf{P}}$	Rate of change of vector \mathbf{P} with respect to frame of fixed orientation
E	Total mechanical energy; voltage	q	Mass rate of flow; electric charge
f	Scalar function	\mathbf{Q}	Force; vector
f_f	Frequency of forced vibration	$\dot{\mathbf{Q}}$	Rate of change of vector \mathbf{Q} with respect to frame of fixed orientation
f_n	Natural frequency	$(\dot{\mathbf{Q}})_{Oxyz}$	Rate of change of vector \mathbf{Q} with respect to frame $Oxyz$
\mathbf{F}	Force; friction force	\mathbf{r}	Position vector
g	Acceleration of gravity	$\mathbf{r}_{B/A}$	Position vector of B relative to A
G	Center of gravity; mass center; constant of gravitation	r	Radius; distance; polar coordinate
h	Angular momentum per unit mass	\mathbf{R}	Resultant force; resultant vector; reaction
\mathbf{H}_O	Angular momentum about point O	R	Radius of earth; resistance
$\dot{\mathbf{H}}_G$	Rate of change of angular momentum \mathbf{H}_G with respect to frame of fixed orientation	\mathbf{s}	Position vector
$(\dot{\mathbf{H}}_G)_{Gxyz}$	Rate of change of angular momentum \mathbf{H}_G with respect to rotating frame $Gxyz$	s	Length of arc
$\mathbf{i}, \mathbf{j}, \mathbf{k}$	Unit vectors along coordinate axes	t	Time; thickness; tangential direction
i	Current	\mathbf{T}	Force
I, I_x, \dots	Moments of inertia	T	Tension; kinetic energy
\bar{I}	Centroidal moment of inertia	\mathbf{u}	Velocity
I_{xy}, \dots	Products of inertia	u	Variable
J	Polar moment of inertia	U	Work
k	Spring constant	U_{1-2}^{NC}	work done by non-conservative forces
k_x, k_y, k_O	Radii of gyration	\mathbf{v}, v	Velocity
		v	Speed
		$\bar{\mathbf{v}}, \bar{v}$	Velocity of mass center
		$\mathbf{v}_{B/A}$	Velocity of B relative to frame in translation with A
		$\mathbf{v}_{P/\mathcal{F}}$	Velocity of P relative to rotating frame \mathcal{F}

\mathbf{V}	Vector product	θ	Angular coordinate; Eulerian angle; angle; polar coordinate
V	Volume; potential energy	μ	Coefficient of friction
w	Load per unit length	ρ	Density; radius of curvature
\mathbf{W}, W	Weight; load	τ	Periodic time
x, y, z	Rectangular coordinates; distances	τ_n	Period of free vibration
$\dot{x}, \dot{y}, \dot{z}$	Time derivatives of coordinates x, y, z	ϕ	Angle of friction; Eulerian angle; phase angle; angle
$\bar{x}, \bar{y}, \bar{z}$	Rectangular coordinates of centroid, center of gravity, or mass center	φ	Phase difference
$\boldsymbol{\alpha}, \alpha$	Angular acceleration	ψ	Eulerian angle
α, β, γ	Angles	$\boldsymbol{\omega}, \omega$	Angular velocity
γ	Specific weight	ω_f	Circular frequency of forced vibration
δ	Elongation	ω_n	Natural circular frequency
ε	Eccentricity of conic section or of orbit	$\boldsymbol{\Omega}$	Angular velocity of frame of reference
$\boldsymbol{\lambda}$	Unit vector along a line		
η	Efficiency		



1

Introduction

The tallest skyscraper in the Western Hemisphere, One World Trade Center is a prominent feature of the New York City skyline. From its foundation to its structural components and mechanical systems, the design and operation of the tower is based on the fundamentals of engineering mechanics.

Introduction

- 1.1 WHAT IS MECHANICS?
- 1.2 FUNDAMENTAL CONCEPTS AND PRINCIPLES
- 1.3 SYSTEMS OF UNITS
- 1.4 CONVERTING BETWEEN TWO SYSTEMS OF UNITS
- 1.5 METHOD OF SOLVING PROBLEMS
- 1.6 NUMERICAL ACCURACY

Objectives

- **Define** the science of mechanics and examine its fundamental principles.
- **Discuss** and compare the International System of Units and U.S. Customary Units.
- **Discuss** how to approach the solution of mechanics problems, and introduce the SMART problem-solving methodology.
- **Examine** factors that govern numerical accuracy in the solution of a mechanics problem.

1.1 What is Mechanics?

Mechanics is defined as the science that describes and predicts the conditions of rest or motion of bodies under the action of forces. It consists of the mechanics of *rigid bodies*, mechanics of *deformable bodies*, and mechanics of *fluids*.

The mechanics of rigid bodies is subdivided into **statics** and **dynamics**. Statics deals with bodies at rest; dynamics deals with bodies in motion. In this text, we assume bodies are perfectly rigid. In fact, actual structures and machines are never absolutely rigid; they deform under the loads to which they are subjected. However, because these deformations are usually small, they do not appreciably affect the conditions of equilibrium or the motion of the structure under consideration. They are important, though, as far as the resistance of the structure to failure is concerned. Deformations are studied in a course in mechanics of materials, which is part of the mechanics of deformable bodies. The third division of mechanics, the mechanics of fluids, is subdivided into the study of *incompressible fluids* and of *compressible fluids*. An important subdivision of the study of incompressible fluids is *hydraulics*, which deals with applications involving water.

Mechanics is a physical science, since it deals with the study of physical phenomena. However, some teachers associate mechanics with mathematics, whereas many others consider it as an engineering subject. Both these views are justified in part. Mechanics is the foundation of most engineering sciences and is an indispensable prerequisite to their study. However, it does not have the *empiricism* found in some engineering sciences, i.e., it does not rely on experience or observation alone. The rigor of mechanics and the emphasis it places on deductive reasoning makes it resemble mathematics. However, mechanics is not an *abstract* or even a *pure* science; it is an *applied* science.

The purpose of mechanics is to explain and predict physical phenomena and thus to lay the foundations for engineering applications. You need to know statics to determine how much force will be exerted on a point in a bridge design and whether the structure can withstand that force. Determining the force a dam needs to withstand from the water in a river requires statics. You need statics to calculate how much weight a crane can lift, how much force a locomotive needs to pull a freight train, or how

much force a circuit board in a computer can withstand. The concepts of dynamics enable you to analyze the flight characteristics of a jet, design a building to resist earthquakes, and mitigate shock and vibration to passengers inside a vehicle. The concepts of dynamics enable you to calculate how much force you need to send a satellite into orbit, accelerate a 200,000-ton cruise ship, or design a toy truck that doesn't break. You will not learn how to do these things in this course, but the ideas and methods you learn here will be the underlying basis for the engineering applications you will learn in your work.

1.2 Fundamental Concepts and Principles

Although the study of mechanics goes back to the time of Aristotle (384–322 B.C.) and Archimedes (287–212 B.C.), not until Newton (1642–1727) did anyone develop a satisfactory formulation of its fundamental principles. These principles were later modified by d'Alembert, Lagrange, and Hamilton. Their validity remained unchallenged until Einstein formulated his **theory of relativity** (1905). Although its limitations have now been recognized, **newtonian mechanics** still remains the basis of today's engineering sciences.

The basic concepts used in mechanics are *space*, *time*, *mass*, and *force*. These concepts cannot be truly defined; they should be accepted on the basis of our intuition and experience and used as a mental frame of reference for our study of mechanics.

The concept of **space** is associated with the position of a point P . We can define the position of P by providing three lengths measured from a certain reference point, or *origin*, in three given directions. These lengths are known as the *coordinates* of P .

To define an event, it is not sufficient to indicate its position in space. We also need to specify the **time** of the event.

We use the concept of **mass** to characterize and compare bodies on the basis of certain fundamental mechanical experiments. Two bodies of the same mass, for example, are attracted by the earth in the same manner; they also offer the same resistance to a change in translational motion.

A **force** represents the action of one body on another. A force can be exerted by actual contact, like a push or a pull, or at a distance, as in the case of gravitational or magnetic forces. A force is characterized by its *point of application*, its *magnitude*, and its *direction*; a force is represented by a *vector* (Sec. 2.1B).

In newtonian mechanics, space, time, and mass are absolute concepts that are independent of each other. (This is not true in **relativistic mechanics**, where the duration of an event depends upon its position and the mass of a body varies with its velocity.) On the other hand, the concept of force is not independent of the other three. Indeed, one of the fundamental principles of newtonian mechanics listed below is that the resultant force acting on a body is related to the mass of the body and to the manner in which its velocity varies with time.

In this text, you will study the conditions of rest or motion of particles and rigid bodies in terms of the four basic concepts we have introduced. By **particle**, we mean a very small amount of matter, which we

assume occupies a single point in space. A **rigid body** consists of a large number of particles occupying fixed positions with respect to one another. The study of the mechanics of particles is clearly a prerequisite to that of rigid bodies. Besides, we can use the results obtained for a particle directly in a large number of problems dealing with the conditions of rest or motion of actual bodies.

The study of elementary mechanics rests on six fundamental principles, based on experimental evidence.

- **The Parallelogram Law for the Addition of Forces.** Two forces acting on a particle may be replaced by a single force, called their *resultant*, obtained by drawing the diagonal of the parallelogram with sides equal to the given forces (Sec. 2.1A).
- **The Principle of Transmissibility.** The conditions of equilibrium or of motion of a rigid body remain unchanged if a force acting at a given point of the rigid body is replaced by a force of the same magnitude and same direction, but acting at a different point, provided that the two forces have the same line of action (Sec. 3.1B).
- **Newton's Three Laws of Motion.** Formulated by Sir Isaac Newton in the late seventeenth century, these laws can be stated as follows:

FIRST LAW. If the resultant force acting on a particle is zero, the particle remains at rest (if originally at rest) or moves with constant speed in a straight line (if originally in motion) (Sec. 2.3B).

SECOND LAW. If the resultant force acting on a particle is not zero, the particle has an acceleration proportional to the magnitude of the resultant and in the direction of this resultant force.

As you will see in Sec. 12.1, this law can be stated as

$$\mathbf{F} = m\mathbf{a} \quad (1.1)$$

where \mathbf{F} , m , and \mathbf{a} represent, respectively, the resultant force acting on the particle, the mass of the particle, and the acceleration of the particle expressed in a consistent system of units.

THIRD LAW. The forces of action and reaction between bodies in contact have the same magnitude, same line of action, and opposite sense (Ch. 6, Introduction).

- **Newton's Law of Gravitation.** Two particles of mass M and m are mutually attracted with equal and opposite forces \mathbf{F} and $-\mathbf{F}$ of magnitude F (Fig. 1.1), given by the formula

$$F = G \frac{Mm}{r^2} \quad (1.2)$$

where r = the distance between the two particles and G = a universal constant called the *constant of gravitation*. Newton's law of gravitation introduces the idea of an action exerted at a distance and extends the range of application of Newton's third law: the action \mathbf{F} and the reaction $-\mathbf{F}$ in Fig. 1.1 are equal and opposite, and they have the same line of action.

A particular case of great importance is that of the attraction of the earth on a particle located on its surface. The force \mathbf{F} exerted by the earth on the particle is defined as the **weight \mathbf{W}** of the particle. Suppose we set

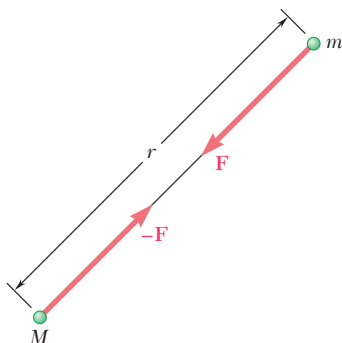


Fig. 1.1 From Newton's law of gravitation, two particles of masses M and m exert forces upon each other of equal magnitude, opposite direction, and the same line of action. This also illustrates Newton's third law of motion.

M equal to the mass of the earth, m equal to the mass of the particle, and r equal to the earth's radius R . Then introducing the constant

$$g = \frac{GM}{R^2} \quad (1.3)$$

we can express the magnitude W of the weight of a particle of mass m as[†]

$$W = mg \quad (1.4)$$

The value of R in formula (1.3) depends upon the elevation of the point considered; it also depends upon its latitude, since the earth is not truly spherical. The value of g therefore varies with the position of the point considered. However, as long as the point actually remains on the earth's surface, it is sufficiently accurate in most engineering computations to assume that g equals 9.81 m/s^2 or 32.2 ft/s^2 .

The principles we have just listed will be introduced in the course of our study of mechanics as they are needed. The statics of particles carried out in Chap. 2 will be based on the parallelogram law of addition and on Newton's first law alone. We introduce the principle of transmissibility in Chap. 3 as we begin the study of the statics of rigid bodies, and we bring in Newton's third law in Chap. 6 as we analyze the forces exerted on each other by the various members forming a structure. We introduce Newton's second law and Newton's law of gravitation in dynamics. We will then show that Newton's first law is a particular case of Newton's second law (Sec. 12.1) and that the principle of transmissibility could be derived from the other principles and thus eliminated (Sec. 16.1D). In the meantime, however, Newton's first and third laws, the parallelogram law of addition, and the principle of transmissibility will provide us with the necessary and sufficient foundation for the entire study of the statics of particles, rigid bodies, and systems of rigid bodies.

As noted earlier, the six fundamental principles listed previously are based on experimental evidence. Except for Newton's first law and the principle of transmissibility, they are independent principles that cannot be derived mathematically from each other or from any other elementary physical principle. On these principles rests most of the intricate structure of newtonian mechanics. For more than two centuries, engineers have solved a tremendous number of problems dealing with the conditions of rest and motion of rigid bodies, deformable bodies, and fluids by applying these fundamental principles. Many of the solutions obtained could be checked experimentally, thus providing a further verification of the principles from which they were derived. Only in the twentieth century has Newton's mechanics found to be at fault, in the study of the motion of atoms and the motion of the planets, where it must be supplemented by the theory of relativity. On the human or engineering scale, however, where velocities are small compared with the speed of light, Newton's mechanics have yet to be disproved.

1.3 Systems of Units

Associated with the four fundamental concepts just discussed are the so-called *kinetic units*, i.e., the units of *length*, *time*, *mass*, and *force*. These units cannot be chosen independently if Eq. (1.1) is to be satisfied.

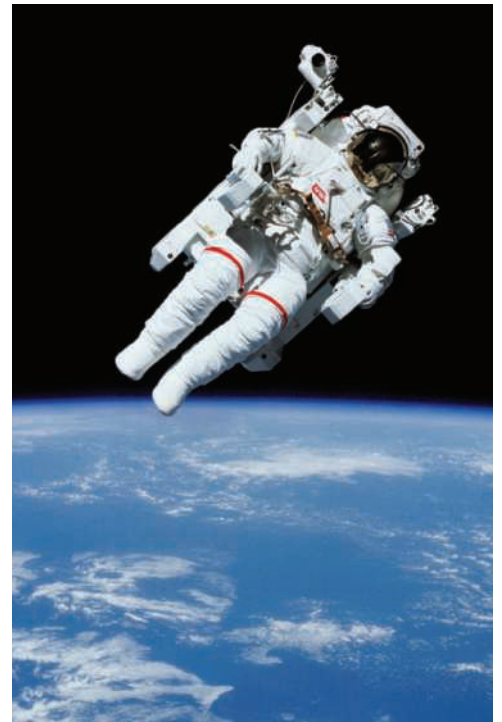


Photo 1.1 When in orbit of the earth, people and objects are said to be *weightless* even though the gravitational force acting is approximately 90% of that experienced on the surface of the earth. This apparent contradiction will be resolved in Chapter 12 when we apply Newton's second law to the motion of particles.

[†]A more accurate definition of the weight \mathbf{W} should take into account the earth's rotation.



Fig. 1.2 A force of 1 newton applied to a body of mass 1 kg provides an acceleration of 1 m/s^2 .

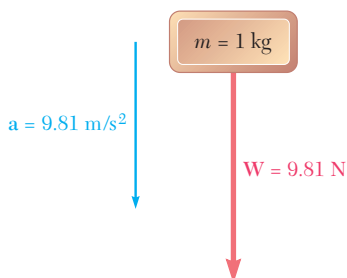


Fig. 1.3 A body of mass 1 kg experiencing an acceleration due to gravity of 9.81 m/s^2 has a weight of 9.81 N.

Three of the units may be defined arbitrarily; we refer to them as **basic units**. The fourth unit, however, must be chosen in accordance with Eq. (1.1) and is referred to as a **derived unit**. Kinetic units selected in this way are said to form a **consistent system of units**.

International System of Units (SI Units).[†] In this system, which will be in universal use after the United States has completed its conversion to SI units, the base units are the units of length, mass, and time, and they are called, respectively, the **meter** (m), the **kilogram** (kg), and the **second** (s). All three are arbitrarily defined. The second was originally chosen to represent $1/86\,400$ of the mean solar day, but it is now defined as the duration of $9\,192\,631\,770$ cycles of the radiation corresponding to the transition between two levels of the fundamental state of the cesium-133 atom. The meter, originally defined as one ten-millionth of the distance from the equator to either pole, is now defined as $1\,650\,763.73$ wavelengths of the orange-red light corresponding to a certain transition in an atom of krypton-86. (The newer definitions are much more precise and with today's modern instrumentation, are easier to verify as a standard.) The kilogram, which is approximately equal to the mass of 0.001 m^3 of water, is defined as the mass of a platinum-iridium standard kept at the International Bureau of Weights and Measures at Sèvres, near Paris, France. The unit of force is a derived unit. It is called the **newton** (N) and is defined as the force that gives an acceleration of 1 m/s^2 to a body of mass 1 kg (Fig. 1.2). From Eq. (1.1), we have

$$1 \text{ N} = (1 \text{ kg})(1 \text{ m/s}^2) = 1 \text{ kg}\cdot\text{m/s}^2 \quad (1.5)$$

The SI units are said to form an *absolute* system of units. This means that the three base units chosen are independent of the location where measurements are made. The meter, the kilogram, and the second may be used anywhere on the earth; they may even be used on another planet and still have the same significance.

The *weight* of a body, or the *force of gravity* exerted on that body, like any other force, should be expressed in newtons. From Eq. (1.4), it follows that the weight of a body of mass 1 kg (Fig. 1.3) is

$$\begin{aligned} W &= mg \\ &= (1 \text{ kg})(9.81 \text{ m/s}^2) \\ &= 9.81 \text{ N} \end{aligned}$$

Multiples and submultiples of the fundamental SI units are denoted through the use of the prefixes defined in Table 1.1. The multiples and submultiples of the units of length, mass, and force most frequently used in engineering are, respectively, the *kilometer* (km) and the *millimeter* (mm); the *megagram*[‡] (Mg) and the *gram* (g); and the *kilonewton* (kN). According to Table 1.1, we have

$$\begin{aligned} 1 \text{ km} &= 1000 \text{ m} & 1 \text{ mm} &= 0.001 \text{ m} \\ 1 \text{ Mg} &= 1000 \text{ kg} & 1 \text{ g} &= 0.001 \text{ kg} \\ 1 \text{ kN} &= 1000 \text{ N} \end{aligned}$$

The conversion of these units into meters, kilograms, and newtons, respectively, can be effected by simply moving the decimal point three places

[†]SI stands for *Système International d'Unités* (French)

[‡]Also known as a *metric ton*.

Table 1.1 SI Prefixes

Multiplication Factor	Prefix [†]	Symbol
1 000 000 000 000 = 10 ¹²	tera	T
1 000 000 000 = 10 ⁹	giga	G
1 000 000 = 10 ⁶	mega	M
1 000 = 10 ³	kilo	k
100 = 10 ²	hecto [‡]	h
10 = 10 ¹	deka [‡]	da
0.1 = 10 ⁻¹	deci [‡]	d
0.01 = 10 ⁻²	centi [‡]	c
0.001 = 10 ⁻³	milli	m
0.000 001 = 10 ⁻⁶	micro	μ
0.000 000 001 = 10 ⁻⁹	nano	n
0.000 000 000 001 = 10 ⁻¹²	pico	p
0.000 000 000 000 001 = 10 ⁻¹⁵	femto	f
0.000 000 000 000 000 001 = 10 ⁻¹⁸	atto	a

[†]The first syllable of every prefix is accented, so that the prefix retains its identity. Thus, the preferred pronunciation of kilometer places the accent on the first syllable, not the second.

[‡]The use of these prefixes should be avoided, except for the measurement of areas and volumes and for the nontechnical use of centimeter, as for body and clothing measurements.

to the right or to the left. For example, to convert 3.82 km into meters, move the decimal point three places to the right:

$$3.82 \text{ km} = 3820 \text{ m}$$

Similarly, to convert 47.2 mm into meters, move the decimal point three places to the left:

$$47.2 \text{ mm} = 0.0472 \text{ m}$$

Using engineering notation, you can also write

$$\begin{aligned} 3.82 \text{ km} &= 3.82 \times 10^3 \text{ m} \\ 47.2 \text{ mm} &= 47.2 \times 10^{-3} \text{ m} \end{aligned}$$

The multiples of the unit of time are the *minute* (min) and the *hour* (h). Since 1 min = 60 s and 1 h = 60 min = 3600 s, these multiples cannot be converted as readily as the others.

By using the appropriate multiple or submultiple of a given unit, you can avoid writing very large or very small numbers. For example, it is usually simpler to write 427.2 km rather than 427 200 m and 2.16 mm rather than 0.002 16 m.[†]

Units of Area and Volume. The unit of area is the *square meter* (m²), which represents the area of a square of side 1 m; the unit of volume is the *cubic meter* (m³), which is equal to the volume of a cube of side 1 m. In order to avoid exceedingly small or large numerical values when computing areas and volumes, we use systems of subunits obtained by respectively squaring and cubing not only the millimeter, but also two intermediate

[†]Note that when more than four digits appear on either side of the decimal point to express a quantity in SI units—as in 427 000 m or 0.002 16 m—use spaces, never commas, to separate the digits into groups of three. This practice avoids confusion with the comma used in place of a decimal point, which is the convention in many countries.

submultiples of the meter: the *decimeter* (dm) and the *centimeter* (cm). By definition,

$$\begin{aligned}1 \text{ dm} &= 0.1 \text{ m} = 10^{-1} \text{ m} \\1 \text{ cm} &= 0.01 \text{ m} = 10^{-2} \text{ m} \\1 \text{ mm} &= 0.001 \text{ m} = 10^{-3} \text{ m}\end{aligned}$$

Therefore, the submultiples of the unit of area are

$$\begin{aligned}1 \text{ dm}^2 &= (1 \text{ dm})^2 = (10^{-1} \text{ m})^2 = 10^{-2} \text{ m}^2 \\1 \text{ cm}^2 &= (1 \text{ cm})^2 = (10^{-2} \text{ m})^2 = 10^{-4} \text{ m}^2 \\1 \text{ mm}^2 &= (1 \text{ mm})^2 = (10^{-3} \text{ m})^2 = 10^{-6} \text{ m}^2\end{aligned}$$

Similarly, the submultiples of the unit of volume are

$$\begin{aligned}1 \text{ dm}^3 &= (1 \text{ dm})^3 = (10^{-1} \text{ m})^3 = 10^{-3} \text{ m}^3 \\1 \text{ cm}^3 &= (1 \text{ cm})^3 = (10^{-2} \text{ m})^3 = 10^{-6} \text{ m}^3 \\1 \text{ mm}^3 &= (1 \text{ mm})^3 = (10^{-3} \text{ m})^3 = 10^{-9} \text{ m}^3\end{aligned}$$

Note that when measuring the volume of a liquid, the cubic decimeter (dm^3) is usually referred to as a *liter* (L).

Table 1.2 shows other derived SI units used to measure the moment of a force, the work of a force, etc. Although we will introduce these units in later chapters as they are needed, we should note an important rule at

Table 1.2 Principal SI Units Used in Mechanics

Quantity	Unit	Symbol	Formula
Acceleration	Meter per second squared	...	m/s^2
Angle	Radian	rad	†
Angular acceleration	Radian per second squared	...	rad/s^2
Angular velocity	Radian per second	...	rad/s
Area	Square meter	...	m^2
Density	Kilogram per cubic meter	...	kg/m^3
Energy	Joule	J	$\text{N}\cdot\text{m}$
Force	Newton	N	$\text{kg}\cdot\text{m/s}^2$
Frequency	Hertz	Hz	s^{-1}
Impulse	Newton-second	...	$\text{kg}\cdot\text{m/s}$
Length	Meter	m	‡
Mass	Kilogram	kg	‡
Moment of a force	Newton-meter	...	$\text{N}\cdot\text{m}$
Power	Watt	W	J/s
Pressure	Pascal	Pa	N/m^2
Stress	Pascal	Pa	N/m^2
Time	Second	s	‡
Velocity	Meter per second	...	m/s
Volume			
Solids	Cubic meter	...	m^3
Liquids	Liter	L	10^{-3} m^3
Work	Joule	J	$\text{N}\cdot\text{m}$

†Supplementary unit (1 revolution = 2π rad = 360°).

‡Base unit.

this time: When a derived unit is obtained by dividing a base unit by another base unit, you may use a prefix in the numerator of the derived unit, but not in its denominator. For example, the constant k of a spring that stretches 20 mm under a load of 100 N is expressed as

$$k = \frac{100 \text{ N}}{20 \text{ mm}} = \frac{100 \text{ N}}{0.020 \text{ m}} = 5000 \text{ N/m} \text{ or } k = 5 \text{ kN/m}$$

but never as $k = 5 \text{ N/mm}$.

U.S. Customary Units. Most practicing American engineers still commonly use a system in which the base units are those of length, force, and time. These units are, respectively, the *foot* (ft), the *pound* (lb), and the *second* (s). The second is the same as the corresponding SI unit. The foot is defined as 0.3048 m. The pound is defined as the *weight* of a platinum standard, called the *standard pound*, which is kept at the National Institute of Standards and Technology outside Washington D.C., the mass of which is 0.453 592 43 kg. Since the weight of a body depends upon the earth's gravitational attraction, which varies with location, the standard pound should be placed at sea level and at a latitude of 45° to properly define a force of 1 lb. Clearly the U.S. customary units do not form an absolute system of units. Because they depend upon the gravitational attraction of the earth, they form a *gravitational* system of units.

Although the standard pound also serves as the unit of mass in commercial transactions in the United States, it cannot be used that way in engineering computations, because such a unit would not be consistent with the base units defined in the preceding paragraph. Indeed, when acted upon by a force of 1 lb—that is, when subjected to the force of gravity—the standard pound has the acceleration due to gravity, $g = 32.2 \text{ ft/s}^2$ (Fig. 1.4), not the unit acceleration required by Eq. (1.1). The unit of mass consistent with the foot, the pound, and the second is the mass that receives an acceleration of 1 ft/s^2 when a force of 1 lb is applied to it (Fig. 1.5). This unit, sometimes called a *slug*, can be derived from the equation $F = ma$ after substituting 1 lb for F and 1 ft/s^2 for a . We have

$$F = ma \quad 1 \text{ lb} = (1 \text{ slug})(1 \text{ ft/s}^2)$$

This gives us

$$1 \text{ slug} = \frac{1 \text{ lb}}{1 \text{ ft/s}^2} = 1 \text{ lb} \cdot \text{s}^2/\text{ft} \quad (1.6)$$

Comparing Figs. 1.4 and 1.5, we conclude that the slug is a mass 32.2 times larger than the mass of the standard pound.

The fact that, in the U.S. customary system of units, bodies are characterized by their weight in pounds rather than by their mass in slugs is convenient in the study of statics, where we constantly deal with weights and other forces and only seldom deal directly with masses. However, in the study of dynamics, where forces, masses, and accelerations are involved, the mass m of a body is expressed in slugs when its weight W is given in pounds. Recalling Eq. (1.4), we write

$$m = \frac{W}{g} \quad (1.7)$$

where g is the acceleration due to gravity ($g = 32.2 \text{ ft/s}^2$).

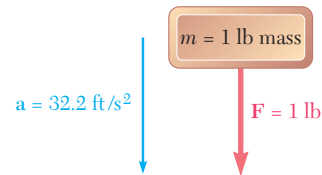


Fig. 1.4 A body of 1 pound mass acted upon by a force of 1 pound has an acceleration of 32.2 ft/s^2 .

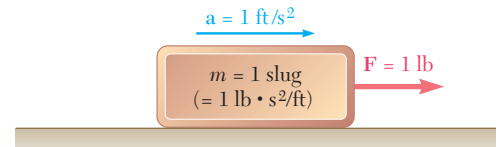


Fig. 1.5 A force of 1 pound applied to a body of mass 1 slug produces an acceleration of 1 ft/s^2 .